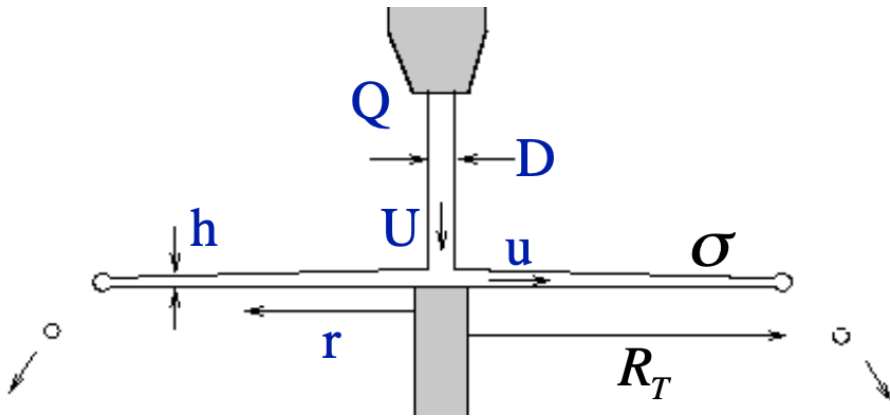


Lecture 13: Fluid sheets, chains and fishbones

Fluid sheets: generated by jets colliding with an impactor or another jet

Circular Sheet

- for $We = \frac{\rho U^2 D}{\sigma} > 10^3$, sheet flaps as on a soap film
- consider case of $We < 10^3$, where a stable circular sheet arises



Scaling: $Re \sim \frac{UR}{\nu} \sim \frac{30 \cdot 10}{0.01} \sim 3 \times 10^4 \gg 1$
 \Rightarrow inertia dominates viscosity

$Fr = \frac{U^2}{gR} \sim \frac{30^2}{10^3 \cdot 10} \sim 0.1 \Rightarrow$ inertia dominates g

Dominant force balance: inertia vs. curvature

$\rho u^2 h = 2\sigma$ ★

Continuity : $Q = 2\pi r u h$

$\Rightarrow h(r) = \frac{Q}{2\pi r u} \sim \frac{1}{r}$ since $u = \text{const}$ (expts)

Subbing $h(r)$ into \star yields sheet radius :

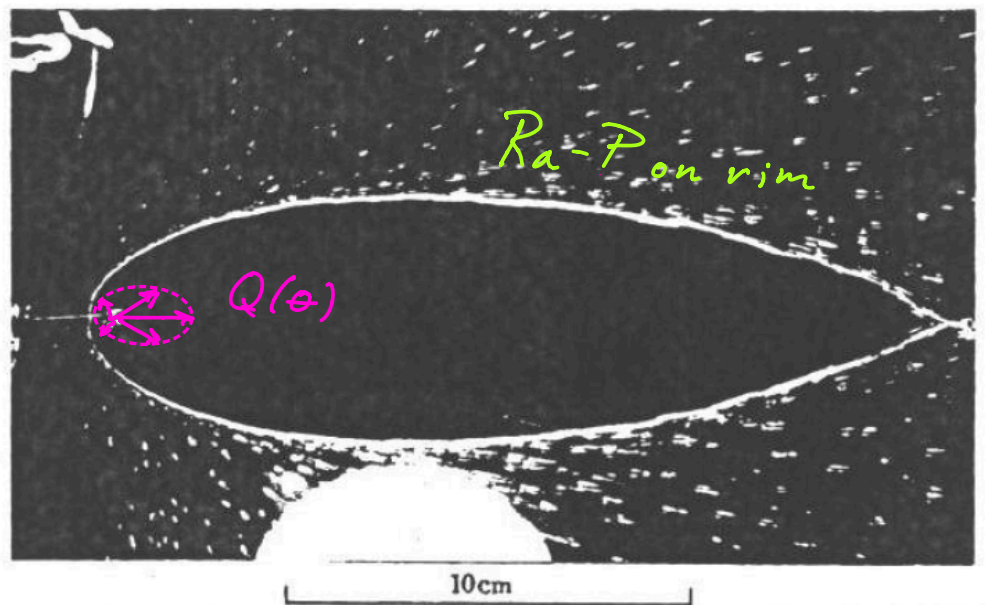
$$R_T = \frac{\rho Q u}{4\pi \sigma}$$

Taylor radius

- Note :
1. mass released from rim via Ra-P
 2. same forms generated by colliding jets
 3. polygonal sheets arise for more viscous sheets

Formation of thin flat sheets of water (Taylor 1960)

Case 1 : Unstable rims (eject droplets)



Sheet shape set by balance of curvature + inertia :

$\rho u_n^2 h \sim 2\sigma$ where $u_n = \text{speed normal to rim}$

Sheet thickness : $h = \frac{Q(\theta)}{2\pi r u}$

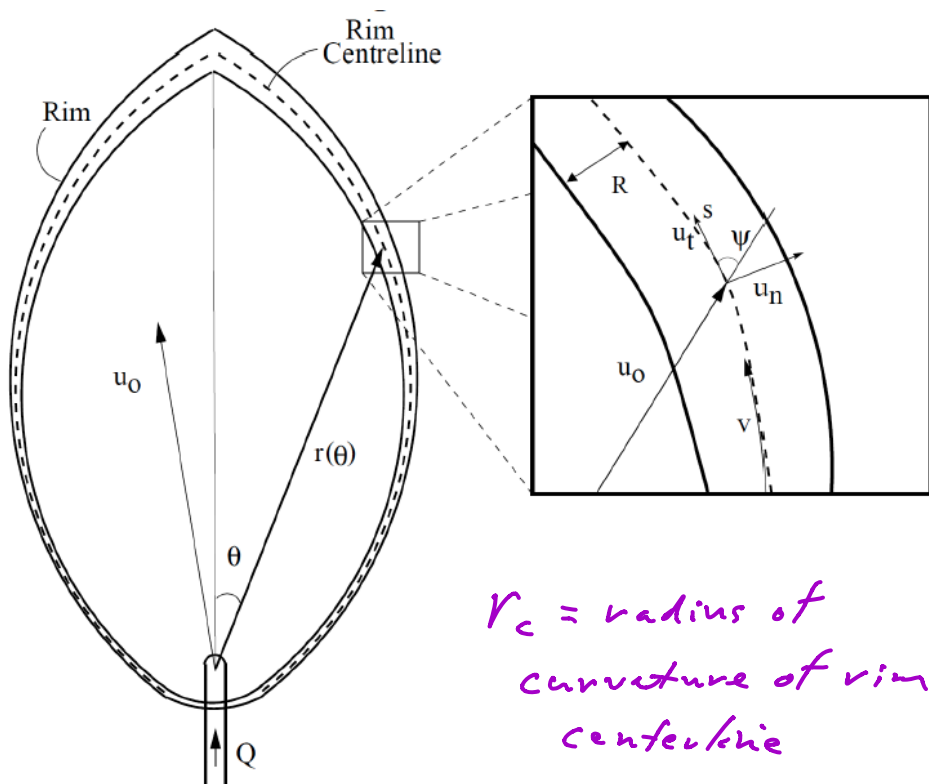
Taylor radius :

$$R_T(\theta) = \frac{\rho u_n^2 Q(\theta)}{4\pi \sigma u}$$

now a
fn of θ

Case 2 : Stable rims

- arise with more viscous fluids
- when rims are stable, fluid enters the rim and proceeds along it
- there is a centripetal force on rim associated with flow along it
 \Rightarrow must be considered in shape calculation



$R_c = \text{radius of curvature of rim centreline}$

Normal Force Balance on Rim:

$$\rho u_n^2 h + \frac{\rho \pi R^2 v^2}{r_c} = 2\sigma$$

INERTIA CENTRIFUGAL CURVATURE

Continuity in Rim:

$$0 = u_n h - \frac{d}{ds} (v \pi R^2)$$

VOL FLUX FROM SHEET TANG. GRADIENT IN VOL FLUX ALONG RIM

Tangential Force Balance

$$\frac{d}{ds} (\pi R^2 v^2) = \rho h u_t + u_n - \pi R^2 \sigma \frac{d}{ds} (v \cdot \frac{1}{r_c}) + 3\mu \frac{d}{ds} (R^2 \frac{dv}{ds})$$

TANG GRAD. IN TANG MOM FLUX TANG MOM FLUX FROM SHEET GRADIENT IN CURVATURE PRESSURE VISCOUS RESISTANCE TO STRETCHING

$\frac{1}{r_c}$ if $R_c \ll 1$

Continuity in sheet: $h(v, \theta) = \frac{Q(\theta)}{u_0 v}$

Geometry: $u_r = u_0 \sin \psi$, $u_t = u_0 \cos \psi$

$$\frac{1}{r_c} = \frac{\sin \psi}{r} \left(\frac{d\psi}{d\theta} + 1 \right)$$

- this system may be nondimensionalized and reduced to a set of coupled ODEs in the four variables $v(\theta)$, $v(\theta)$, $R(\theta)$, $\psi(\theta)$.
- given $Q(\theta)$, these may be integrated to deduce the sheet shape

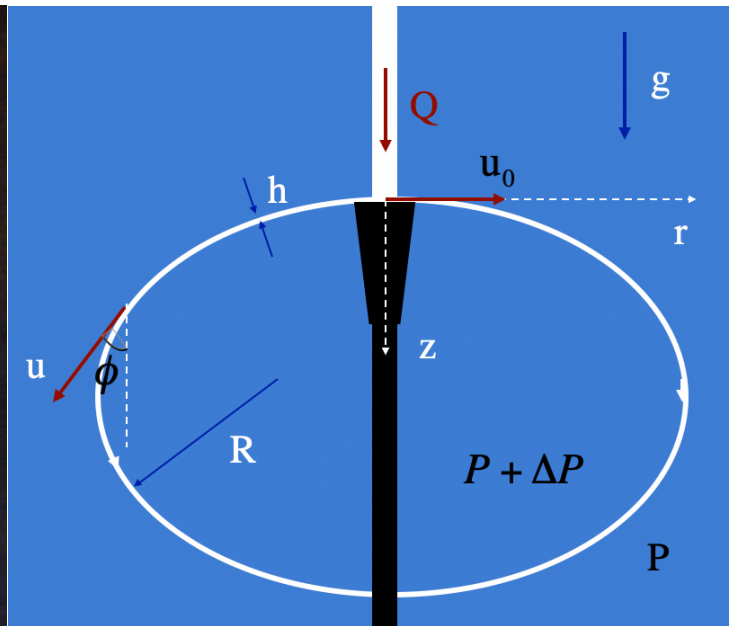
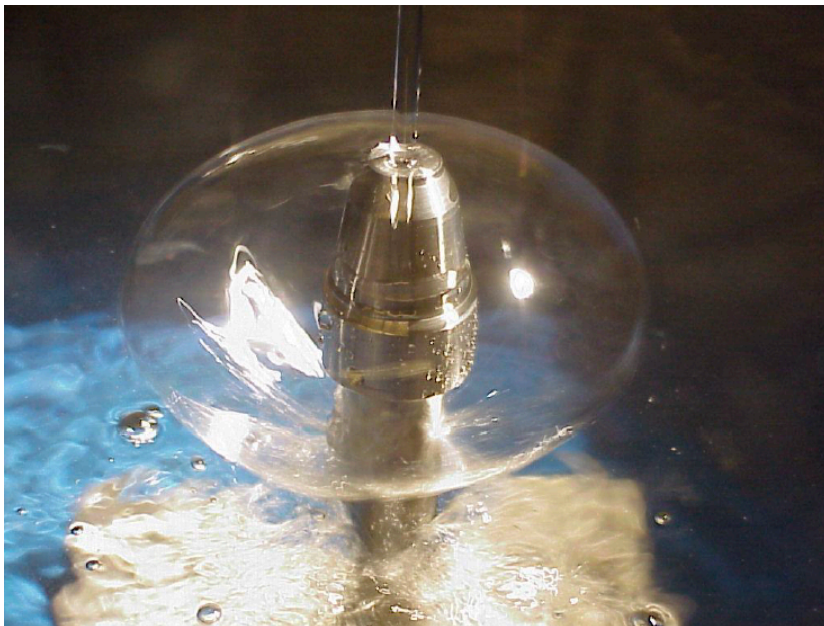
Fluid Fishbones

- arise when timescale of $Ra-P$ of the rim is comparable to time spent in the rim
- $Ra-P$ sets in \Rightarrow bulbous regions hung outwards \Rightarrow "fishbones"
- bones pinch off via $Ra-P$



Water Balls (Savart 1833, Taylor 1959, Clenet)

- gravity causes sheets to sag \Rightarrow curvature causes sheet to close



Bell Shape : prescribed by balance of inertia,
gravity + curvature

Continuity : $Q = 2\pi r h(u)$

Energetic conservation : $u^2 = 2gz + u_0^2$

Sheet curvature : $\nabla \cdot \underline{u} = \underbrace{\frac{1}{R}}_{\text{IN PLANE}} + \underbrace{\frac{\cos \phi}{r}}_{\text{OUT-OF-PLANE}}$

Normal force balance :

$$\underbrace{\frac{2\sigma}{R}}_{\text{CURVATURE}} + \underbrace{\frac{2\sigma \cos \phi}{r}}_{\text{GRAVITY}} + \rho g h \sin \phi = \Delta P + \underbrace{\frac{\rho h u^2}{R}}_{\text{INERTIA}}$$

Note : 1. bell closes due to out-of-plane curvature

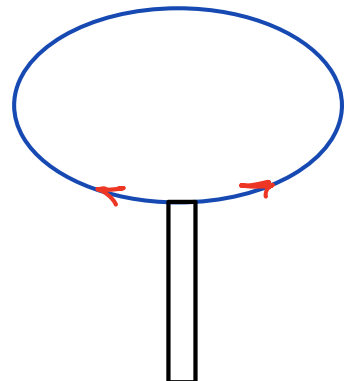
2. influence of g reflected in top-bottom asymmetry

Recall : $F_r = \frac{\text{INERTIA}}{\text{GRAVITY}} \sim \frac{v^2}{gL} \sim 1$

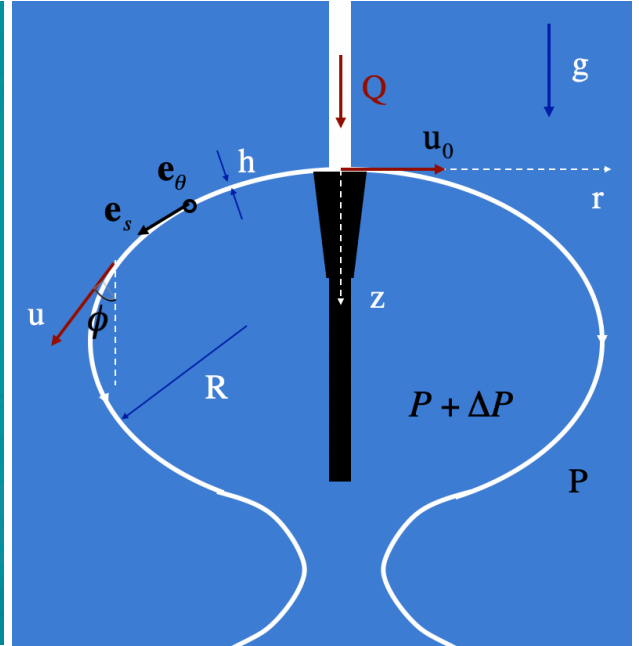
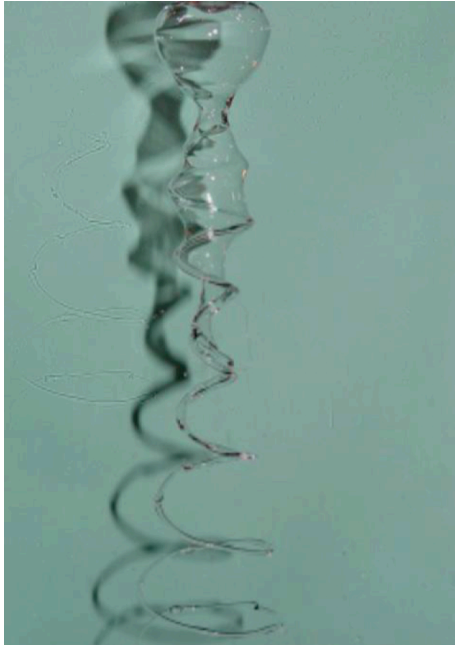
3. if $F_r \gg 1$ and sheet deflected upwards

\Rightarrow bell will still close on itself

4. viscous effect cause various
exotica



Swirling Water Bells : formed from a swirling jet



Why don't they close?

Sheet velocity : $\underline{v} = u \hat{e}_s + \underset{\text{SWIRL}}{V} \hat{e}_\theta$

Continuity : $Q = 2\pi r h u$

Conservation of Ang. Mom : $Vv = V_0 v_0$

Energy Conservation : $u^2 + V^2 = 2gz + u_0^2 + V_0^2$

Normal Force Balance :

$$\frac{2\sigma}{R} + \frac{2\sigma \cos\phi}{v^2} + \rho g h \sin\phi = \Delta P + \frac{\rho h u^2}{R} + \rho \frac{h V^2 \cos\phi}{v}$$

CENTRIFUGAL

\Rightarrow bell fails to close owing to centrifugal force becoming large as $v \rightarrow 0$