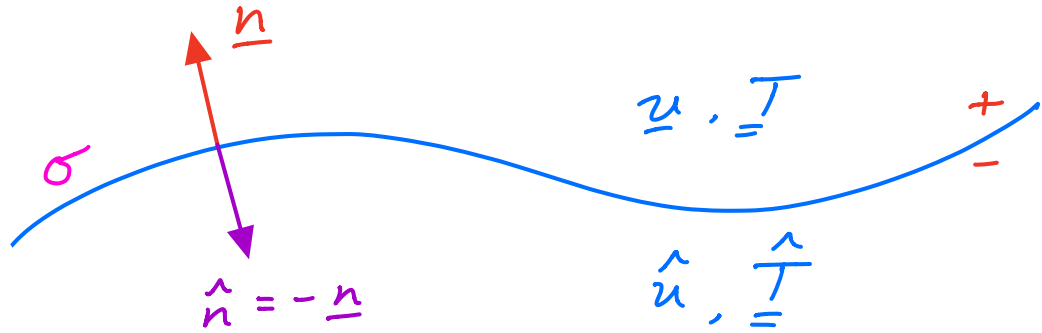


Lecture 5. Interfacial boundary conditions, Statics



⇒ Interfacial Stress Balance Equation

$$\underline{n} \cdot \underline{T} - \underline{\hat{n}} \cdot \underline{\hat{T}} = \sigma \underline{n} \nabla \cdot \underline{n} - \nabla \sigma$$

Stress applied
by + on interface

Stress
applied by
- on interface

NORMAL CURVATURE
PRESSURE ASSOCIATED
WITH $\nabla \cdot \underline{n} = 0$

TANGENTIAL/
MARANGONI
STRESS

Normal Stress Balance

Taking $\underline{n} \cdot \star$ yields

$$\underline{n} \cdot \underline{T} \cdot \underline{n} - \underline{\hat{n}} \cdot \underline{\hat{T}} \cdot \underline{\hat{n}} = \underbrace{\sigma \nabla \cdot \underline{n}}_{\text{LAPLACE PRESSURE}}$$

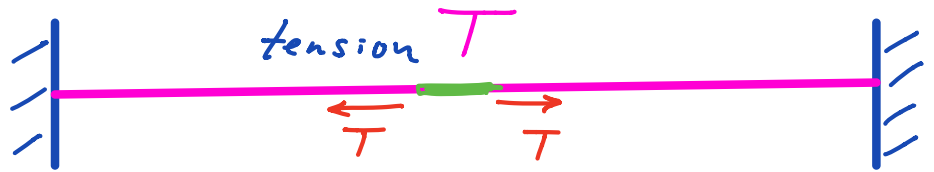
JUMP IN NORMAL STRESS

⇒ DEPENDS ON BOTH

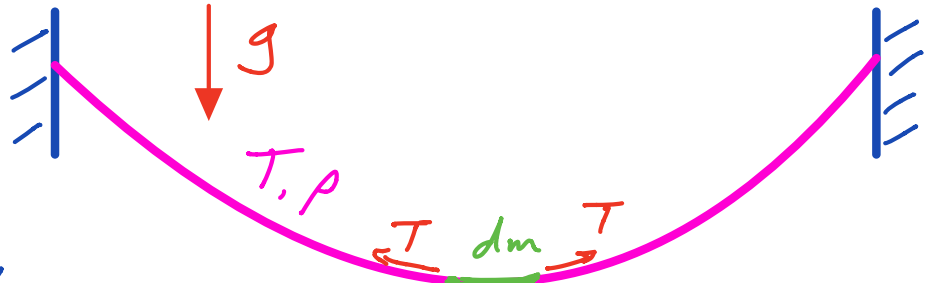
\underline{n}, p

Curvature Forces : An aside...

String under tension



String under tension sags due to gravity



⇒ wt supported by curvature force

Tangential Stress Balance

Taking \underline{t} ★, where \underline{t} is ANY vector tangent to the interface:

$$\underbrace{\underline{n} \cdot \underline{T} \cdot \underline{t} - \underline{n} \cdot \hat{\underline{T}} \cdot \underline{t}}_{\text{JUMP IN TANGENTIAL STRESS}} = \underbrace{\underline{\nabla} \sigma \cdot \underline{t}}_{\text{MARANGONI STRESS ASSOC. w } \underline{\nabla} \sigma}$$

Note: $\underline{\nabla} \sigma$ may arise since $\sigma(C, T)$

\uparrow chemistry \uparrow temperature

⇒ Marangoni stresses arise from $\underline{\nabla} C$ or $\underline{\nabla} T$

∴ the LHS contains only velocity gradients, not pressure; therefore, a non-zero $\underline{\nabla} \sigma$ at a fluid interface ALWAYS drives motion.

Quick Review on Computing Normals

Normal to $z = f(x, y)$

Define functional:

$$g(x, y, z) = z - f(x, y) \\ = 0 \text{ on } S$$

$$\therefore \vec{\nabla} g = \vec{\nabla} (z - f(x, y)) \\ = -f_x \hat{i} - f_y \hat{j} + \hat{k}$$

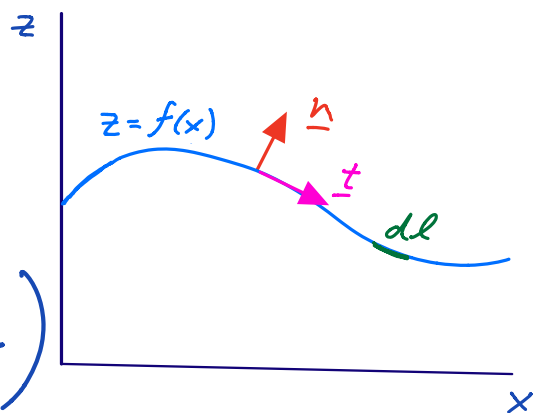
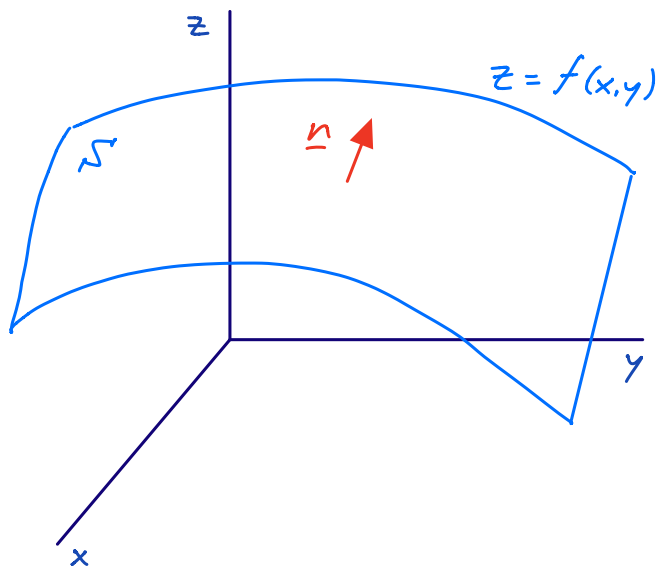
$$\text{and } \hat{n} = \frac{\vec{\nabla} g}{|\vec{\nabla} g|} = \frac{\hat{k} - f_x \hat{i} - f_y \hat{j}}{\sqrt{1 + f_x^2 + f_y^2}}$$

and curvature deduced from $\nabla \cdot \hat{n}$.

Special Case: normal to curve $z = f(x)$

$$\Rightarrow \underline{n} = \frac{\hat{k} - f_x \hat{i}}{\sqrt{1 + f_x^2}}$$

$$\underline{\nabla} \cdot \underline{n} = \left(\hat{i} \frac{d}{dx} + \hat{k} \frac{d}{dz} \right) \cdot \left(\frac{\hat{k} - f_x \hat{i}}{\sqrt{1 + f_x^2}} \right) \\ = \frac{-f_{xx}}{(1 + f_x^2)^{3/2}}$$



An Aside: differential geometry provides some useful relations between \underline{n} and \underline{t} .

$$(\underline{\nabla} \cdot \underline{n}) \underline{n} = \frac{dt}{dl}$$

PROPORTIONAL TO CURVATURE PRESSURE AT INTERFACE

$$-(\underline{\nabla} \cdot \underline{n}) \underline{t} = \frac{dn}{dl}$$

Fluid Statics

- the stress tensor reduces to $\underline{T} = -p \underline{I}$, so that $\underline{n} \cdot \underline{T} \cdot \underline{n} = -p$ and the normal stress balance:

$$\underbrace{\hat{p} - p}_{\Delta p} = \underbrace{\sigma \underline{\nabla} \cdot \underline{n}}_{\text{LAPLACE PRESSURE}}$$

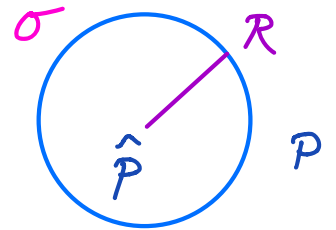
- tangential stress must take the form $\underline{\nabla} \underline{\sigma} = 0$.
 \Rightarrow THERE CAN'T BE A STATIC SYSTEM in the presence of $\underline{\nabla} \underline{\sigma} \neq 0$

Let's consider now a couple of static systems.

I. Stationary Bubble

Recall in spherical coords

$$\underline{\nabla} \cdot \underline{F} = \frac{1}{r^2} \frac{d}{dr} r^2 F_r + \frac{1}{r \sin \theta} \frac{d}{d\theta} (\sin \theta F_\theta) + \frac{1}{r \sin \theta} \frac{d}{d\phi} F_\phi$$



Here $\underline{F} = \underline{n} = (1, 0, 0)$ in (r, θ, ϕ) coords

$$\Rightarrow \underline{\nabla} \cdot \underline{n} = \frac{1}{r^2} \frac{d}{dr} (r^2) = \frac{2}{R}$$

$$\Rightarrow \Delta p = \hat{p} - p = \frac{2\sigma}{R} \quad \text{as previously}$$

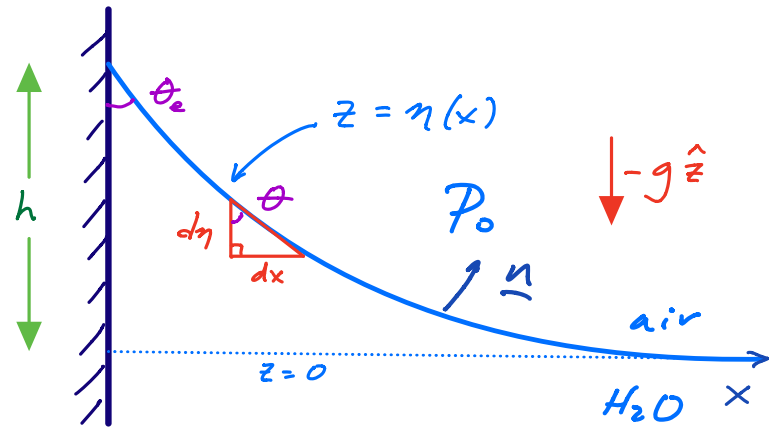
II. The Static Meniscus

Pressure varies according to $p(z) = p_0 + \rho g z$.

Normal stress balance:

$$p_0 - \rho g \eta(x) = p_0 + \sigma \nabla \cdot \underline{n}$$

$$\Rightarrow -\rho g \eta(x) = \sigma \nabla \cdot \underline{n} \quad \boxtimes$$



$$\text{Now } \nabla \cdot \underline{n} = \frac{-\eta_{xx}}{(1 + \eta_x^2)^{3/2}}$$

Approximate sol'n: if slope is everywhere small ($\eta_x \ll 1$), we obtain $(1 + \eta_x^2)^{3/2} \approx 1$

$$\rho g \eta = \sigma \eta_{xx}$$

Apply BCs: $\eta(\infty) = 0$ and $\eta_x(0) = -\cot \theta_e$

$$\text{Solving: } \eta(x) = l_c \cot \theta_e e^{-x/l_c}$$

where $l_c = \sqrt{\sigma/\rho g}$ is capillary note

Note: meniscus drops off on a scale of l_c .

But let's now solve \boxtimes exactly.

We can integrate directly

$$\rho g \eta \eta_x = \sigma \frac{\eta_x \eta_{xx}}{(1 + \eta_x^2)^{3/2}}$$

$$\frac{d}{dx} \frac{\eta^2}{2} = l_c^2 \frac{d}{dx} \frac{1}{(1 + \eta_x^2)^{1/2}}$$

Integrate from x to ∞ :

$$\text{LHS} = \int_x^\infty \frac{d}{dx} \frac{\eta^2}{2} dx = \frac{1}{2} [\eta^2(\infty) - \eta^2(x)] = -\frac{\eta^2}{2}$$

$$\text{RHS} = \int_x^\infty \frac{d}{dx} \frac{1}{(1 + \eta_x^2)^{1/2}} dx = \frac{1}{(1 + 0)^{1/2}} - \frac{1}{(1 + \eta_x^2)^{1/2}} = 1 - \sin\theta$$

$$\Rightarrow \boxed{\sigma \sin\theta + \frac{1}{2} \rho g \eta^2 = \sigma} \quad \star$$

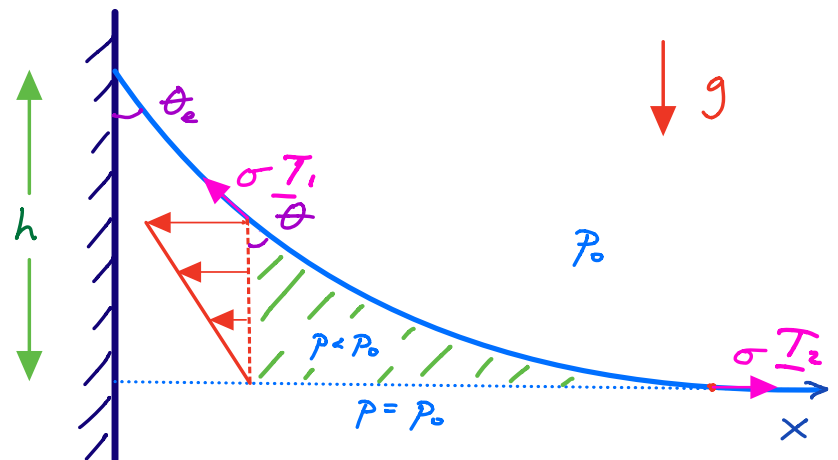
Note: at $z = h$, $\theta = \theta_e \Rightarrow \frac{1}{2} \rho g h^2 = \sigma(1 - \sin\theta_e)$

$$\Rightarrow \boxed{h = \sqrt{2} l_c (1 - \sin\theta_e)^{1/2}}$$

is the maximum rise height

Consider the force balance on a portion of the meniscus.

Horizontal Force Balance



$$\underbrace{\sigma \sin \theta}_{\text{horizontal projection of } \sigma T_1} + \underbrace{\frac{1}{2} \rho g \eta^2}_{\text{hydrostatic suction}} = \underbrace{\sigma}_{\text{tensile stress from } \sigma T_2} \quad \star$$

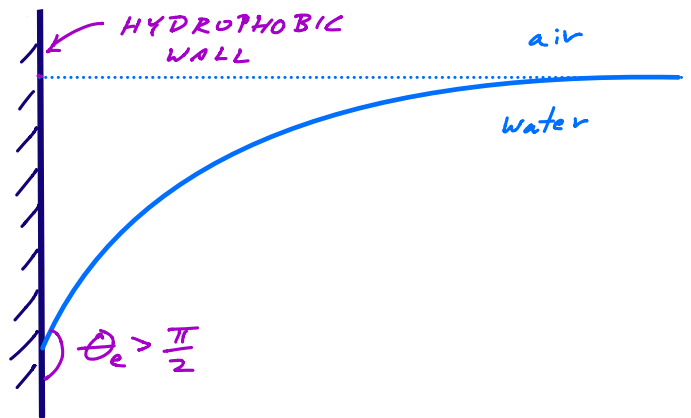
Vertical Force Balance

$$\underbrace{\sigma \cos \theta}_{\text{vertical proj. of } \sigma T_1} = \underbrace{\int_x^\infty \rho g z dx}_{\text{wt of the fluid}}$$

The latter makes clear that, setting $x=0$ (where $\theta=\theta_e$)

$$\Rightarrow \sigma \cos \theta_e = \text{wt of fluid displaced above } z=0$$

\Rightarrow changes sign according to whether $\theta_e > \frac{\pi}{2}$ or $\theta_e < \frac{\pi}{2}$



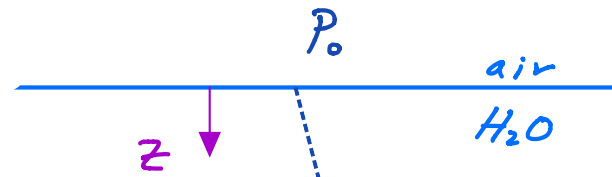
Floating Bodies

"Heavy things sink. Light things float."

\Rightarrow Not really true for small objects.

Recall: Archimedean force on a submerged body.

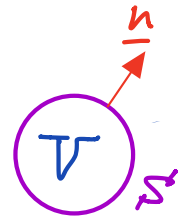
In general, the hydrodynamic force acting on a body in a fluid:



$$\underline{F}_h = \int_S \underline{T} \cdot \underline{n} dS'$$

$$P(z) = p_0 + \rho g z$$

where $\underline{T} = -p \underline{I} + 2\mu \underline{E}$
 $= -p \underline{I}$ for statics



$$\begin{aligned} \Rightarrow \underline{F}_h &= - \int_S p \underline{n} dS' = - \int_S \rho g z \underline{n} dS' \\ &= - \rho g \int_V \nabla z dV \quad \text{by Gen Div Thm} \\ &= - \rho g \int_V dV \hat{z} = - \rho g V \hat{z} \\ &= \text{wt of displaced fluid} \end{aligned}$$

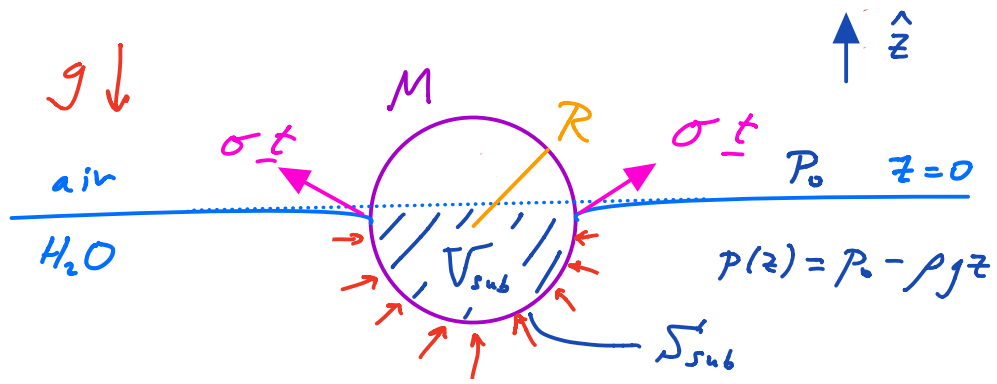
Archimedes Principle: buoyant force is equal to the wt of the displaced fluid

Generalization to Floating Bodies

Case 1: Large bodies ($R \gg l_c$)

Force balance $Mg = \int_{S_{\text{sub}}} \rho g \underline{n} \cdot \hat{z} dS' + 2\pi R \sigma \underline{t} \cdot \hat{z}$

BUOYANCY
SURFACE TENSION



$$\Rightarrow M g = \rho g V_{sub} + 2\pi R \sigma_t \cdot \hat{z}$$

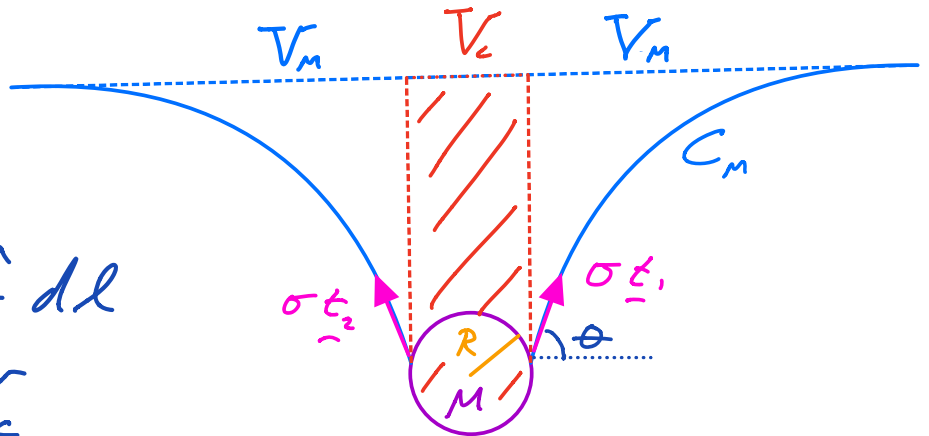
$$\frac{\text{BUOYANCY}}{\text{CAPILLARITY}} \sim \frac{\rho g V_{sub}}{2\pi R \sigma} \sim \frac{\rho g R^3}{R \sigma} \sim \frac{R^2}{l_c^2} = B_0$$

where Bond number $B_0 = \frac{\rho g R^2}{\sigma}$

$\Rightarrow \sigma$ negligible provided $R \gg l_c$

Case 2: Small bodies ($R \leq l_c$)

Vertical force balance: $M g = F_b + F_c$



Buoyancy:

$$F_b = \int_C \rho g z \kappa \cdot \hat{z} dl$$

$$= \rho g V_c$$

where V_c = volume above body, but inside the contact line

Capillary: $F_c = 2\sigma \sin\theta$