

## Lec. 6 : Interfacial flows

- the study of fluid systems dominated by the influence of surface tension at interfaces

Preliminaries : what is surface tension, and what is an interface?

⇒ SLIDES 1-13

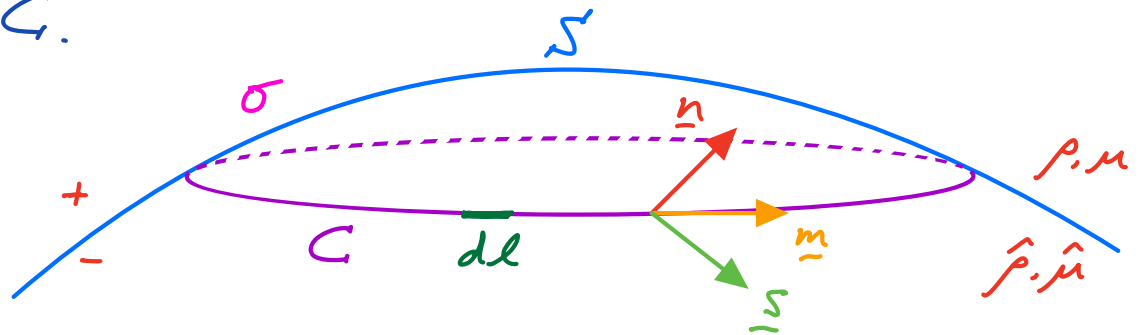
Surface tension : an energy/area at the interface between 2 fluids (liquid-liquid or liquid-gas), of molecular origins that serves to resist the creation of new interface

⇒ imparts a tensile force per unit length to the interface

Deduction of the interfacial stress B.C.s requires consideration of the influence of  $\sigma$  on curved interfaces.

# Interfacial Boundary Conditions

Consider an interfacial surface bound by a closed curve  $C$ .

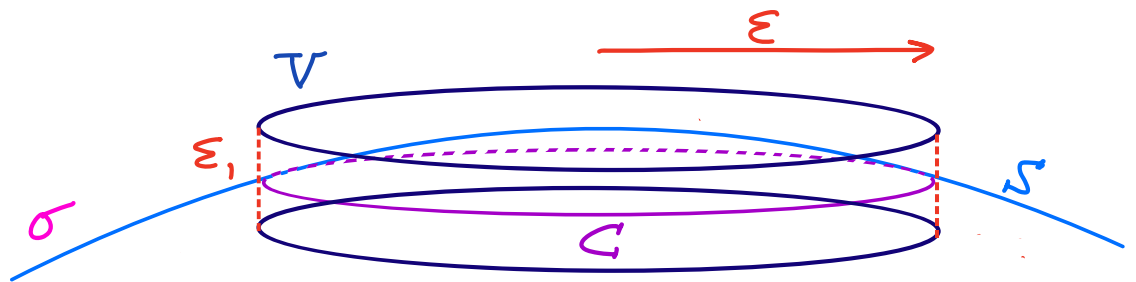


where  $\underline{n}$  is unit normal to  $S$ ,  $C$

$\underline{s}$  is unit normal to  $C$ , tangent to  $S$

$\underline{m}$  is unit tangent to  $S$ ,  $C$

Consider an infinitesimal cylindrical pillbox  $V$  of radius  $\Sigma$  and height  $\varepsilon$ , (s.t.  $\Sigma, \ll \varepsilon$ ) that intersects  $C$ .



For a force balance on  $V$ , we must consider:

$\underline{t}(\underline{n}) = \underline{n} \cdot \underline{T}$  is the force/area exerted by upper fluid (+)

$\hat{\underline{t}}(\hat{\underline{n}}) = \hat{\underline{n}} \cdot \hat{\underline{T}}$  is " " lower fluid (-)

Force balance:

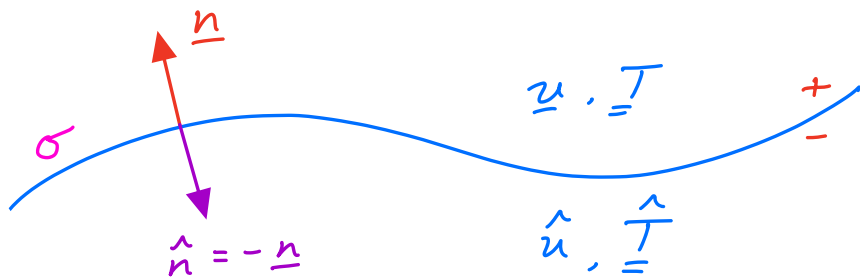
$$\int_V \rho \frac{D\underline{u}}{Dt} dV = \int_V \underline{f} dV + \int_S \left[ \underline{t}(\underline{n}) + \hat{\underline{t}}(\hat{\underline{n}}) \right] dS' + \int_C \sigma \underline{s} dl$$

inertial force
body forces
hydrodynamic forces acting on upper + lower surfaces
surface tension

Now, the inertial + body forces must scale as  $\Sigma^2 \Sigma$ , while the surface forces scale as  $\Sigma^2$ . Hence, in the limit of  $\Sigma_i \rightarrow 0$ , surface forces must balance:

$$\int_S \underline{t}(\underline{n}) + \hat{\underline{t}}(\hat{\underline{n}}) dS' + \int_C \sigma \underline{s} dl = 0$$

Now, we have  $\underline{t}(\underline{n}) = \underline{n} \cdot \underline{T}$ ,  $\hat{\underline{t}}(\hat{\underline{n}}) = \hat{\underline{n}} \cdot \hat{\underline{T}} = -\underline{n} \cdot \hat{\underline{T}}$



Moreover, application of Stokes' Theorem yields

$$\int_C \sigma \underline{s} dl = \int_S \underline{\nabla}_s \sigma - \sigma \underline{n} (\underline{\nabla}_s \cdot \underline{n}) dS'$$

where  $\underline{\nabla}_s \equiv (\underline{T} - \underline{n} \underline{n}) \cdot \underline{\nabla} = \underline{\nabla} - \underline{n} \frac{d}{dn}$

is the tangential gradient operator,

required because  $\sigma$  and  $\underline{n}$  are only defined on the interface. We proceed by dropping the subscript,  $\underline{\nabla}_s \rightarrow \underline{\nabla}$ , with this understanding.

The surface force balance:

$$\int_{\mathcal{S}} (\underline{n} \cdot \underline{T} - \underline{n} \cdot \underline{T}^{\wedge}) d\mathcal{S} = \int_{\mathcal{S}} \sigma \underline{n} (\underline{\nabla} \cdot \underline{n}) - \underline{\nabla} \sigma d\mathcal{S}$$

Now since the surface element  $\mathcal{S}$  is arbitrary, the integrand must vanish identically.

$\Rightarrow$  Interfacial Stress Balance Equation

$$\underline{n} \cdot \underline{T} - \underline{n} \cdot \underline{T}^{\wedge} = \sigma \underline{n} \underline{\nabla} \cdot \underline{n} - \underline{\nabla} \sigma$$

STRESS JUMP AT  
INTERFACE

NORMAL LAPLACE/  
CURVATURE PRESSURE

MARANGONI  
STRESS

Normal Stress Balance

Taking  $\underline{n} \cdot \star$  yields

$$\underline{n} \cdot \underline{T} \cdot \underline{n} - \underline{n} \cdot \underline{T}^{\wedge} \cdot \underline{n} = \sigma \underline{\nabla} \cdot \underline{n}$$

JUMP IN NORMAL STRESS

LAPLACE PRESSURE

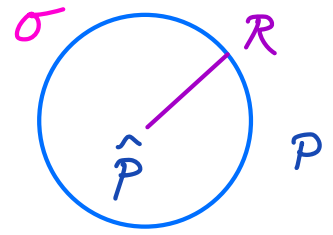
$\Rightarrow$  DEPENDS ON BOTH  $\underline{n}$ ,  $p$



# I. Stationary Bubble

Recall in spherical coords

$$\nabla \cdot \vec{F} = \frac{1}{r^2} \frac{d}{dr} r^2 F_r + \frac{1}{r \sin \theta} \frac{d}{d\theta} (\sin \theta F_\theta) + \frac{1}{r \sin \theta} \frac{d}{d\phi} F_\phi$$



Here  $\vec{F} = \underline{n} = (1, 0, 0)$  in  $(r, \theta, \phi)$  coords

$$\Rightarrow \nabla \cdot \underline{n} = \frac{1}{r^2} \frac{d}{dr} (r^2) = \frac{2}{R}$$

$$\Rightarrow \Delta p = \hat{P} - P = \frac{2\sigma}{R} \Rightarrow \text{smaller bubbles louder}$$

# II. The Static Meniscus

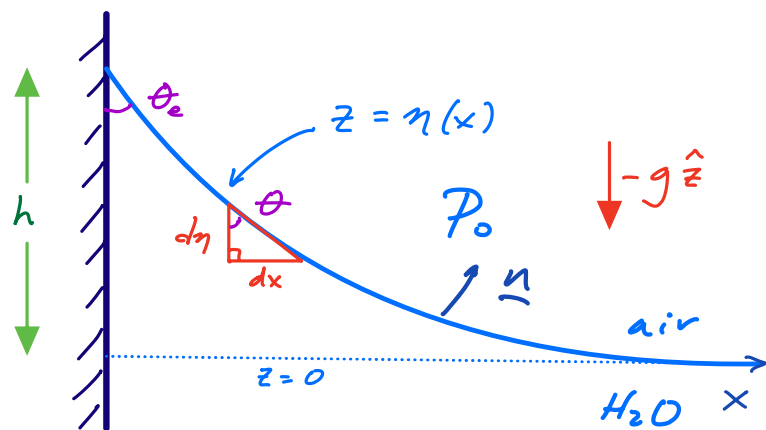
Pressure varies according to  $p(z) = P_0 + \rho g z$ .

Normal stress balance:

$$P_0 - \rho g \eta(x) = P_0 + \sigma \nabla \cdot \underline{n}$$

$$\Rightarrow -\rho g \eta(x) = \sigma \nabla \cdot \underline{n}$$

$$\text{Now } \nabla \cdot \underline{n} = \frac{-\eta_{xx}}{(1 + \eta_x^2)^{3/2}}$$



We can integrate directly

$$\rho g \eta \eta_x = \sigma \frac{\eta_x \eta_{xx}}{(1 + \eta_x^2)^{3/2}}$$

$$\frac{d}{dx} \frac{\eta^2}{2} = l_c^2 \frac{d}{dx} \frac{1}{(1 + \eta_x^2)^{1/2}}$$

Integrate from  $x$  to  $\infty$  :

$$\text{LHS} = \int_x^\infty \frac{d}{dx} \frac{\eta^2}{2} dx = \frac{1}{2} [\eta^2(\infty) - \eta^2(x)] = -\frac{\eta^2}{2}$$

$$\text{RHS} = \int_x^\infty \frac{d}{dx} \frac{1}{(1 + \eta_x^2)^{1/2}} dx = \frac{1}{(1 + 0)^{1/2}} - \frac{1}{(1 + \eta_x^2)^{1/2}} = 1 - \sin\theta$$

$$\Rightarrow \boxed{\sigma \sin\theta + \frac{1}{2} \rho g \eta^2 = \sigma} \quad \star$$

Note: at  $z = h$ ,  $\theta = \theta_e \Rightarrow \frac{1}{2} \rho g h^2 = \sigma (1 - \sin\theta_e)$

$$\Rightarrow \boxed{h = \sqrt{2} l_c (1 - \sin\theta_e)^{1/2}}$$

is the maximum rise height

When is surface tension important?

Indicated by scaling:  $We$ ,  $Bo$ ,  $Ca$

$\Rightarrow$  SLIDES

---

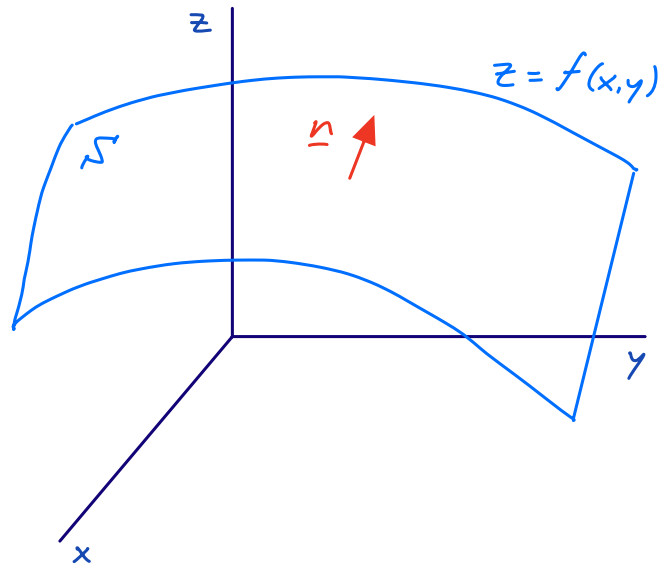
Bonus/Supplementary Material not in lecture 6...

# Quick Review on Computing Normals

## Normal to $z = f(x, y)$

Define functional:

$$g(x, y, z) = z - f(x, y) \\ = 0 \text{ on } S$$



$$\therefore \vec{\nabla} g = \vec{\nabla} (z - f(x, y)) \\ = -f_x \hat{i} - f_y \hat{j} + \hat{k}$$

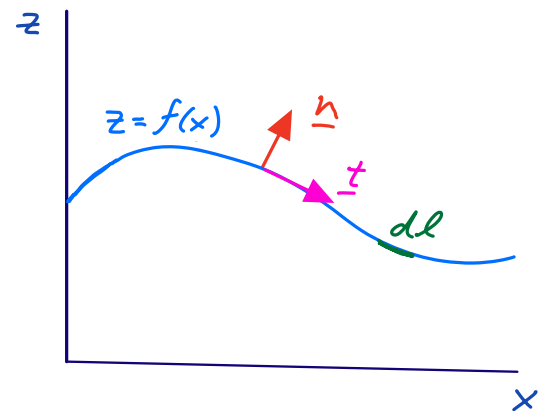
$$\text{and } \hat{n} = \frac{\vec{\nabla} g}{|\vec{\nabla} g|} = \frac{\hat{k} - f_x \hat{i} - f_y \hat{j}}{\sqrt{1 + f_x^2 + f_y^2}}$$

and curvature deduced from  $\nabla \cdot \hat{n}$ .

Special case: normal to curve  $z = f(x)$

$$\Rightarrow \hat{n} = \frac{\hat{k} - f_x \hat{i}}{\sqrt{1 + f_x^2}}$$

$$\nabla \cdot \hat{n} = \left( \hat{i} \frac{d}{dx} + \hat{k} \frac{d}{dz} \right) \cdot \left( \frac{\hat{k} - f_x \hat{i}}{\sqrt{1 + f_x^2}} \right) \\ = \frac{-f_{xx}}{(1 + f_x^2)^{3/2}}$$



## Floating Bodies

"Heavy things sink. Light things float."

$\Rightarrow$  Not really true for small objects.



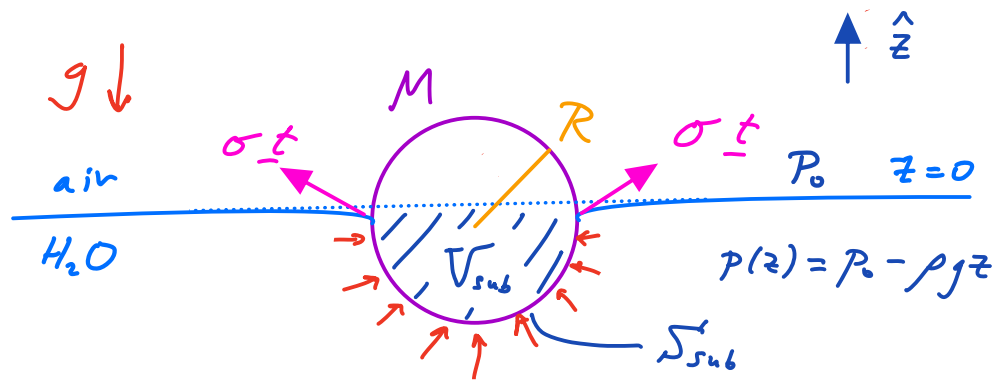
Archimedes Principle: buoyant force is equal to the wt of the displaced fluid

## Generalization to Floating Bodies

Case 1: Large bodies ( $R \gg l_c$ )

Force balance  $Mg = \int_{\mathcal{V}_{sub}} \rho g \underline{n} \cdot \hat{\underline{z}} dV + 2\pi R \sigma \underline{t} \cdot \hat{\underline{z}}$

BUOYANCY
SURFACE TENSION



$$\Rightarrow Mg = \rho g V_{sub} + 2\pi R \sigma \underline{t} \cdot \hat{\underline{z}}$$

$$\frac{\text{BUOYANCY}}{\text{CAPILLARITY}} \sim \frac{\rho g V_{sub}}{2\pi R \sigma} \sim \frac{\rho g R^3}{R \sigma} \sim \frac{R^2}{l_c^2} = B_0$$

where Bond number  $B_0 = \frac{\rho g R^2}{\sigma}$

$\Rightarrow \sigma$  negligible provided  $R \gg l_c$

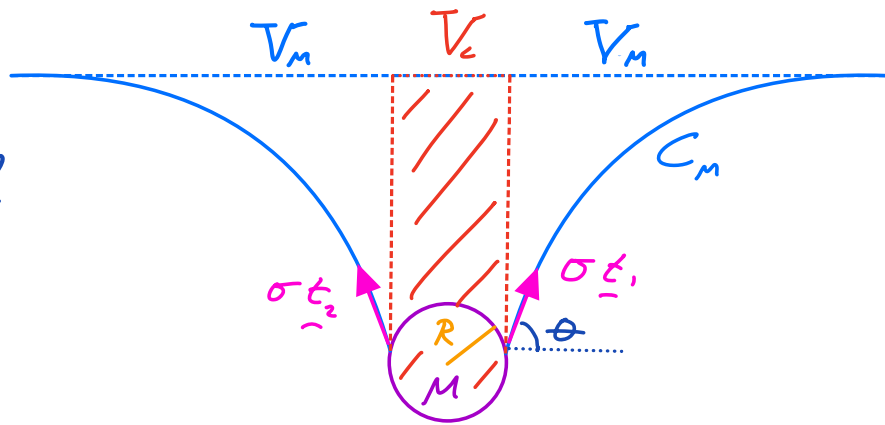
Case 2: Small bodies ( $R \leq l_c$ )

Vertical force balance:  $Mg = F_b + F_c$

Buoyancy :

$$F_b = \int_C \rho g z \underline{n} \cdot \hat{z} dl$$

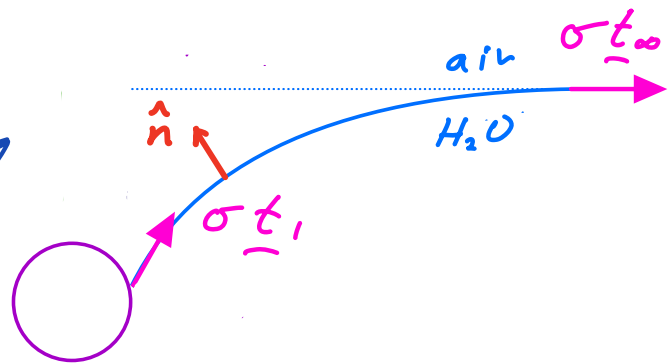
$$= \rho g V_c$$



where  $V_c$  = volume above body, but inside the contact line

Capillary :  $F_c = 2\sigma \sin\theta$

Now, pressure balance along the meniscus  $C_m$  :



$\rho g z \underline{n} = \sigma \underline{\nabla} \cdot \underline{n} \underline{n} \Rightarrow$  integrate along  $C_m$

$\Rightarrow \int_{C_m} \rho g z \underline{n} dl = \int_{C_m} \sigma \underline{\nabla} \cdot \underline{n} \underline{n} dl$

$\frac{dt}{dl}$  via Frenet-Serret

$\Rightarrow \rho g V_m = \sigma (t_2 - t_1) \cdot \hat{z} = \sigma \sin\theta$

$\Rightarrow 2\sigma \sin\theta = 2\rho g V_m$

CAP FORCE WT DISPLACED OUTSIDE CONTACT LINE by MENISCUS

Archimedes Principle is still valid!